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# Basic 5

## Micro-actuator

### - Electrostatic actuator -

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**Nagoya University**



# Micro-actuator

## Electrostatic force

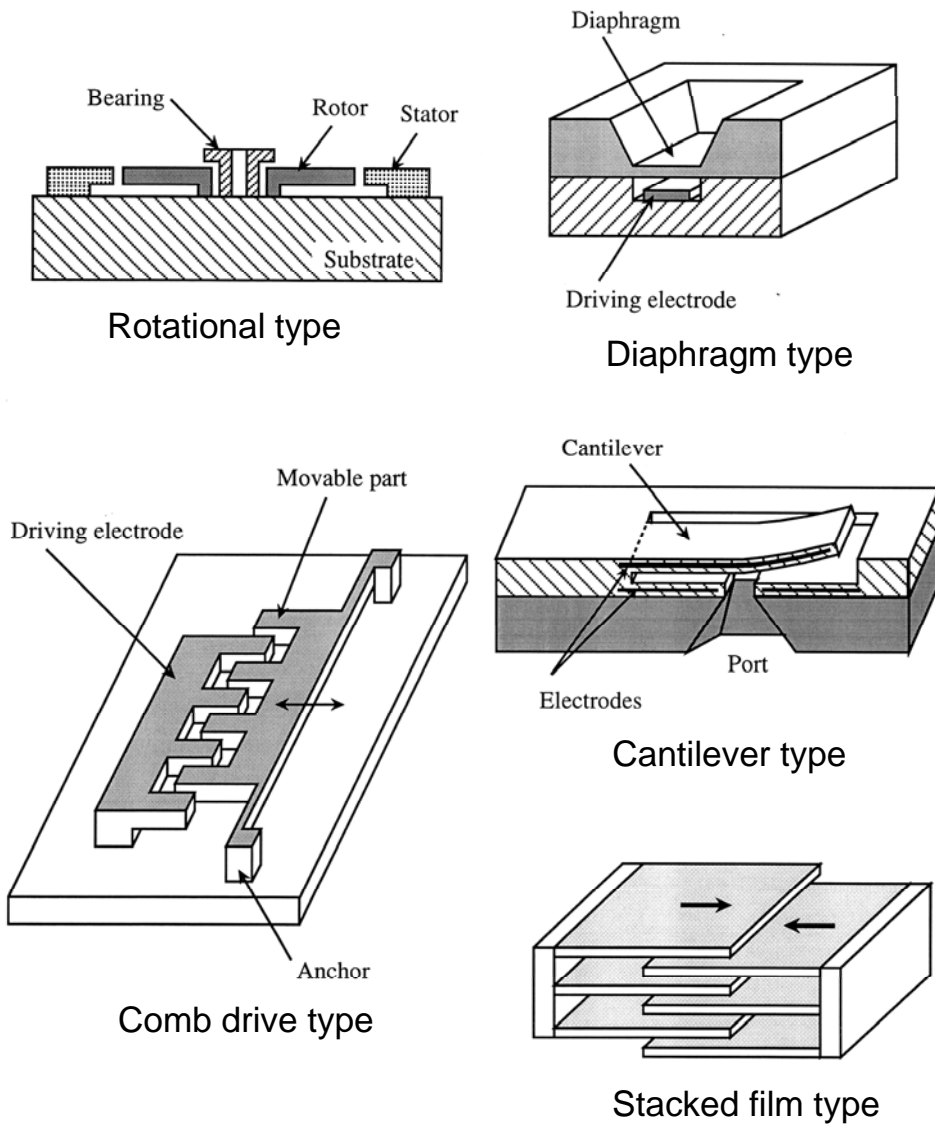


図 2-2. 静電力駆動アクチュエータのデバイス構造

## Magnetic force

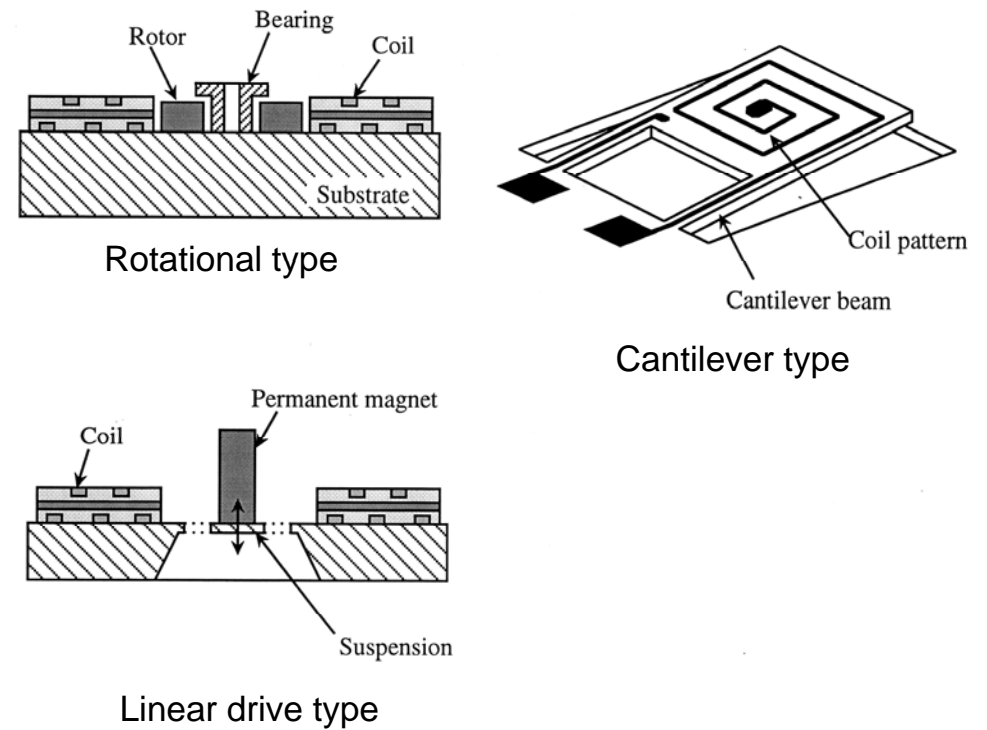
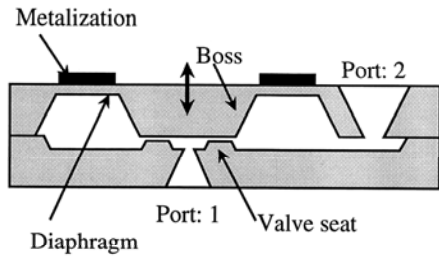


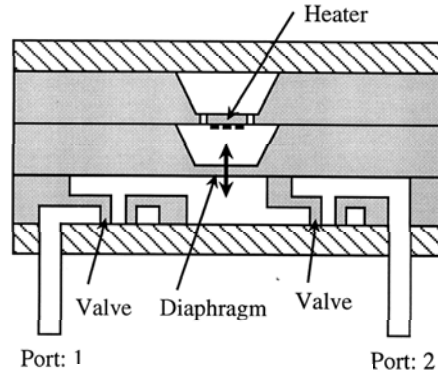
図 2-3. 電磁力駆動アクチュエータのデバイス構造

# Micro-actuator

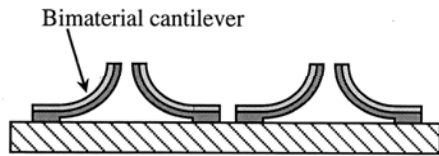
## Thermal force



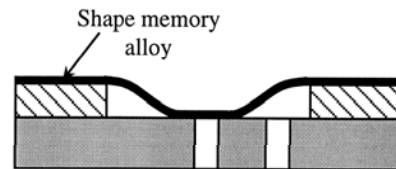
Bi-metal  
(Diaphragm type)



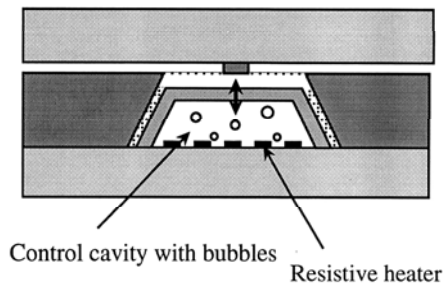
Pneumatic type



Bi-metal  
(Cantilever type)



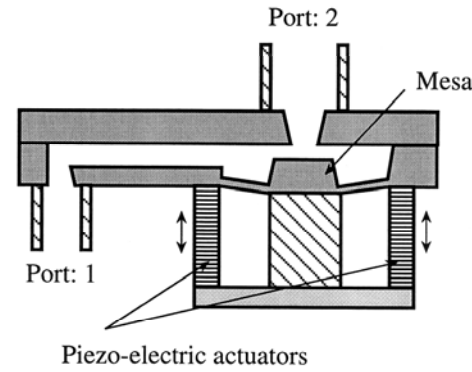
SMA type



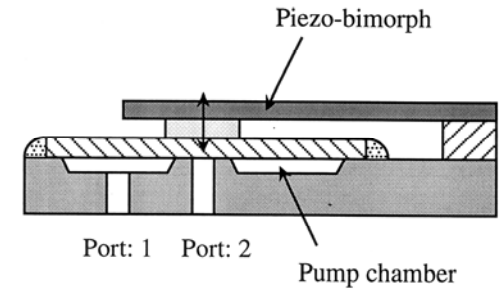
Thermal expansion type

図 2-5. 熱駆動アクチュエータのデバイス構造

## Piezoelectric force



Diaphragm type



Cantilever type

図 2-4. 圧電駆動アクチュエータのデバイス構造

# Micro-actuator

## Comparison of various force applied in Micro-actuator

	Force	Response	Power consumption	Integration
Electrostatic force	+	+++	+++	+++
Magnetic force	+++	+++	+	+
Thermal force	+++	+	+	++
Piezo electric force (Stack type)	++	+++	++	+
Piezo electric force (Film type)	+	+++	+++	+++



# ***Electrostatic actuator***

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***The reason why electrostatic force is applied in micro-sized actuator***

- ✓ Scale effect***
- ✓ Simple structure***
- ✓ Compatible with MEMS fabrication process***
- ✓ Integration with IC and micro-sensor***



# Generative force

## Comparison between electrostatic and magnetic forces (Force per unit area)

$$U_E = \frac{\varepsilon}{2} E^2,$$

$$U_M = \frac{\mu}{2} H^2,$$

$\varepsilon$ : Dielectric constant, E: Electrical field  
 $\mu$ : Permeability, H: Magnetic field

$$E = 3 \times 10^6 \text{ (V / m)}$$

$$B = 1 \text{ (T)}$$

$$U_E \approx 10^1 \text{ (N / m}^2\text{)}$$

$$U_M \approx 10^5 \text{ (N / m}^2\text{)}$$

# Generative force

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## *Scale effect*

$$U_E = \frac{\varepsilon}{2} E^2, \quad \varepsilon: \text{Dielectric constant, } E: \text{Electrical field}$$



$$F_E = \frac{\varepsilon}{2} k E^2 S, \quad k: \text{Constant, } S: \text{Surface area}$$



$$F_E \propto L^2 \quad (@ E : \text{const})$$



# Generative force

## Scale effect

$$U_M = \frac{\mu}{2} H^2, \quad \mu: \text{Permeability, } H: \text{Magnetic field}$$

→  $F_M \propto L^{2\sim 3}$   
*(In the case of permanent magnet)*

→  $F_M \propto L^{3\sim 4}$   
*(In the case of electric magnet)*



# Electrostatic force

## Electrostatic energy

$$U_E = \frac{1}{2} QV = \frac{1}{2} \frac{Q^2}{C} = \frac{1}{2} CV^2,$$

*C: Capacitance, Q: Charge,  
V: Potential*

## Electrostatic force

$$F_x = - \left( \frac{\partial U_E}{\partial x} \right)_{Q=const} \quad \leftarrow \text{In the case of } Q=\text{constant}$$

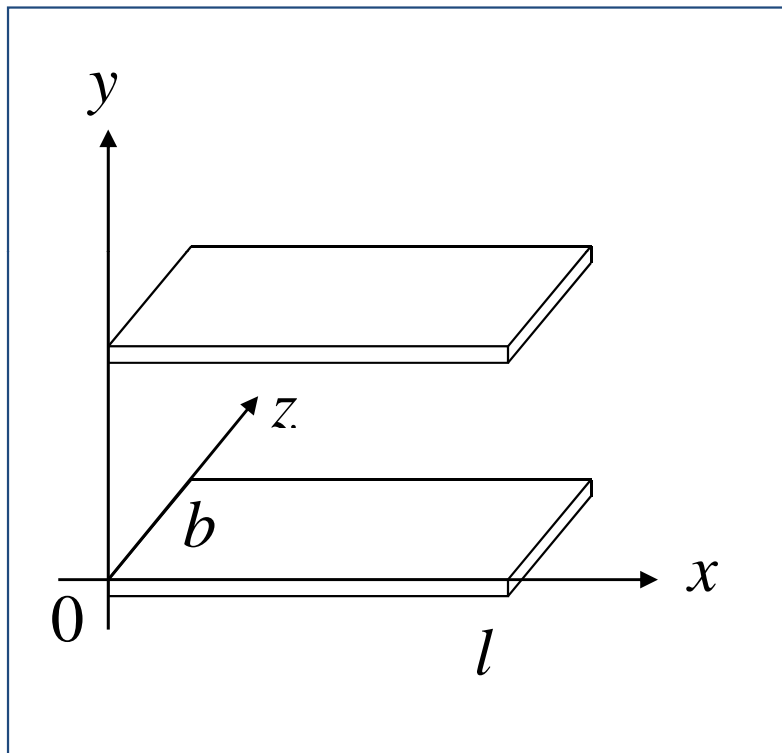
$$F_x = \left( \frac{\partial U_E}{\partial x} \right)_{V=const} \quad \leftarrow \text{In the case of } V=\text{constant}$$

*Energy is supplied by electrical power source*

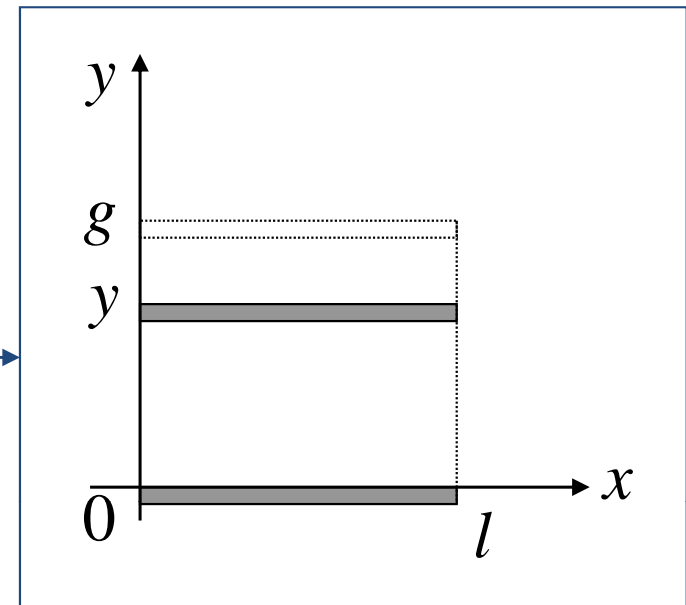


# Electrostatic force

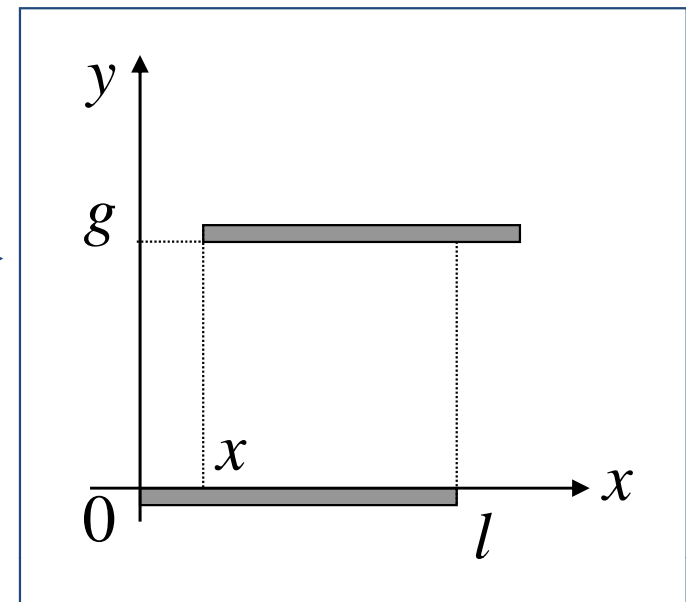
## Parallel plate type



*Normal force*

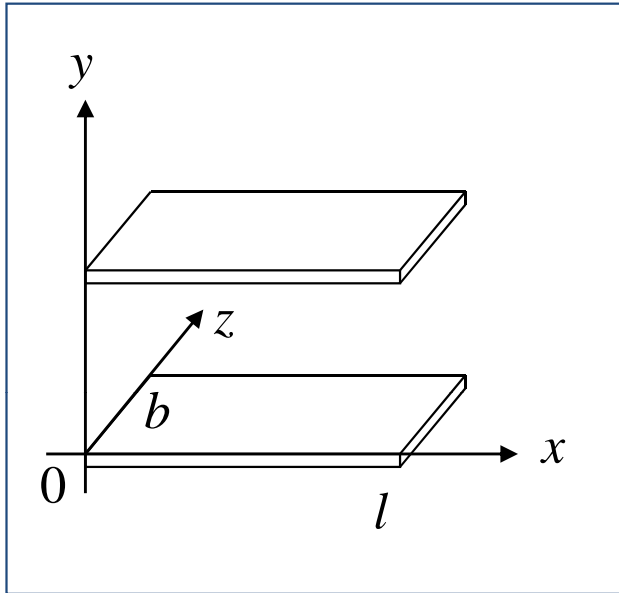


*Lateral force*



# Electrostatic force

## Normal component force



$$C = \epsilon \frac{lb}{y}$$

$$U_E = \frac{y}{2\epsilon lb} Q^2 = \frac{\epsilon lb}{2y} V^2$$

Force

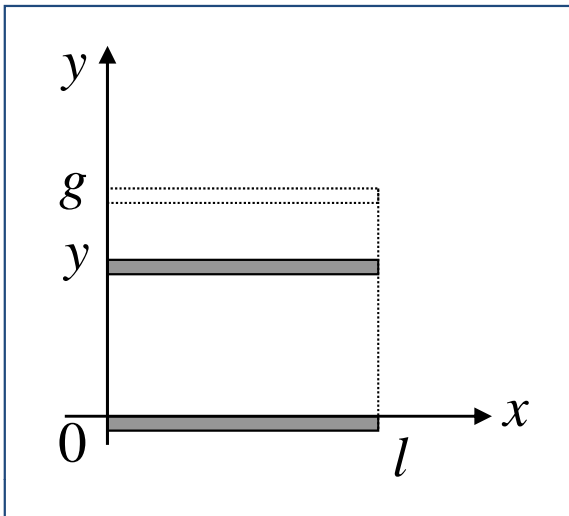
[ $Q = \text{const.}$ ]

$$F_y = - \left( \frac{\partial U_E}{\partial y} \right)_{Q=\text{const}} = - \frac{Q^2}{2\epsilon lb}$$

[ $V = \text{const.}$ ]

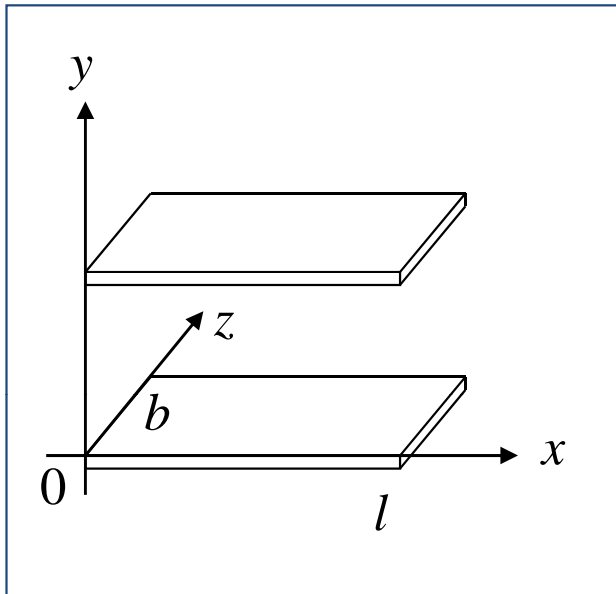
$$F_y = \left( \frac{\partial U_E}{\partial y} \right)_{V=\text{const}} = - \frac{\epsilon lb V^2}{2y^2}$$

*Force is inverse square function of distance*



# Electrostatic force

## Lateral component force



$$C = \epsilon \frac{(l-x)b}{g}$$

$$U_E = \frac{g}{2\epsilon(l-x)b} Q^2 = \frac{\epsilon(l-x)b}{2g} V^2$$

Force

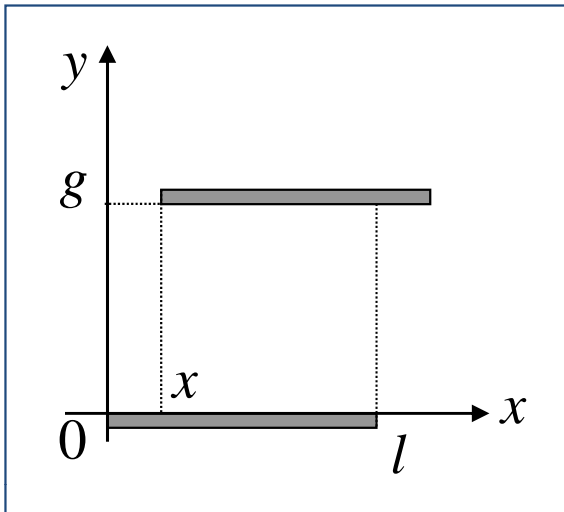
[ $Q = \text{const.}$ ]

$$F_x = - \left( \frac{\partial U_E}{\partial x} \right)_{Q=\text{const}} = - \frac{g}{2\epsilon(l-x)^2 b} Q^2$$

[ $V = \text{const.}$ ]

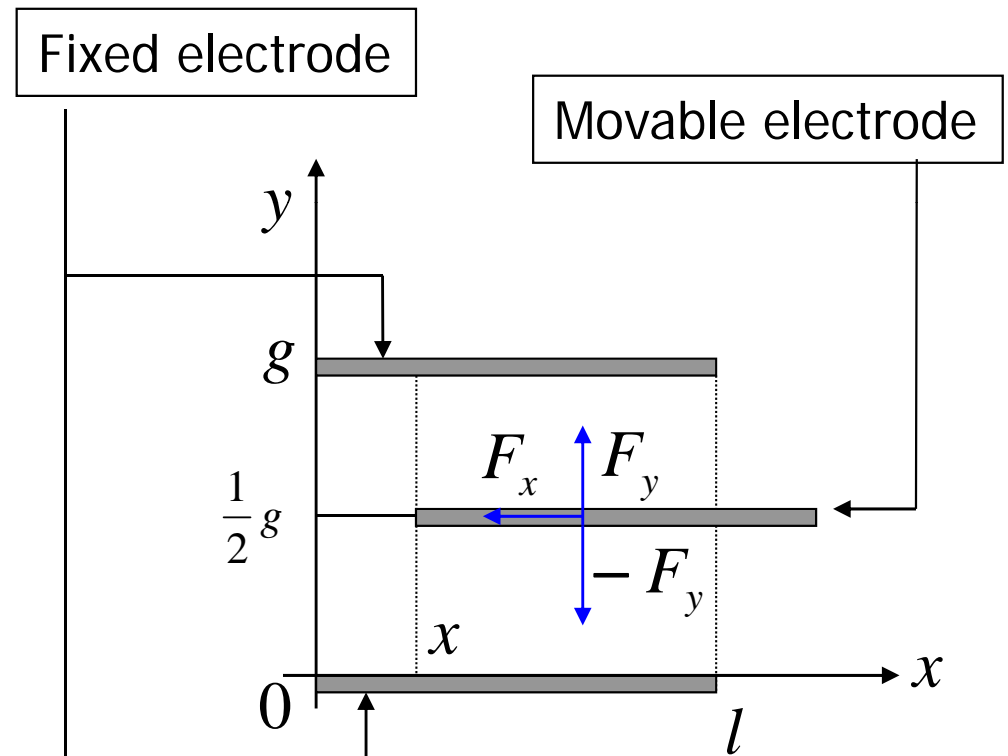
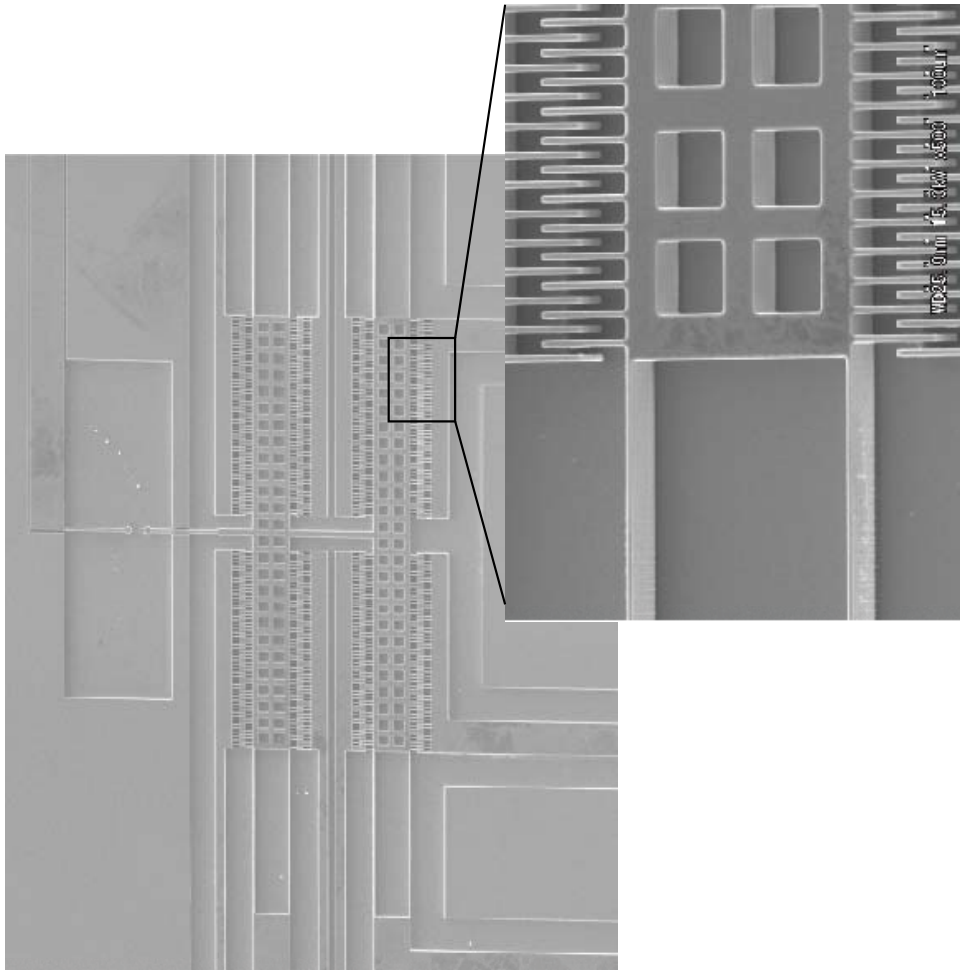
$$F_x = \left( \frac{\partial U_E}{\partial x} \right)_{V=\text{const}} = - \frac{\epsilon b}{2g} V^2$$

**Force does not depend on distance**



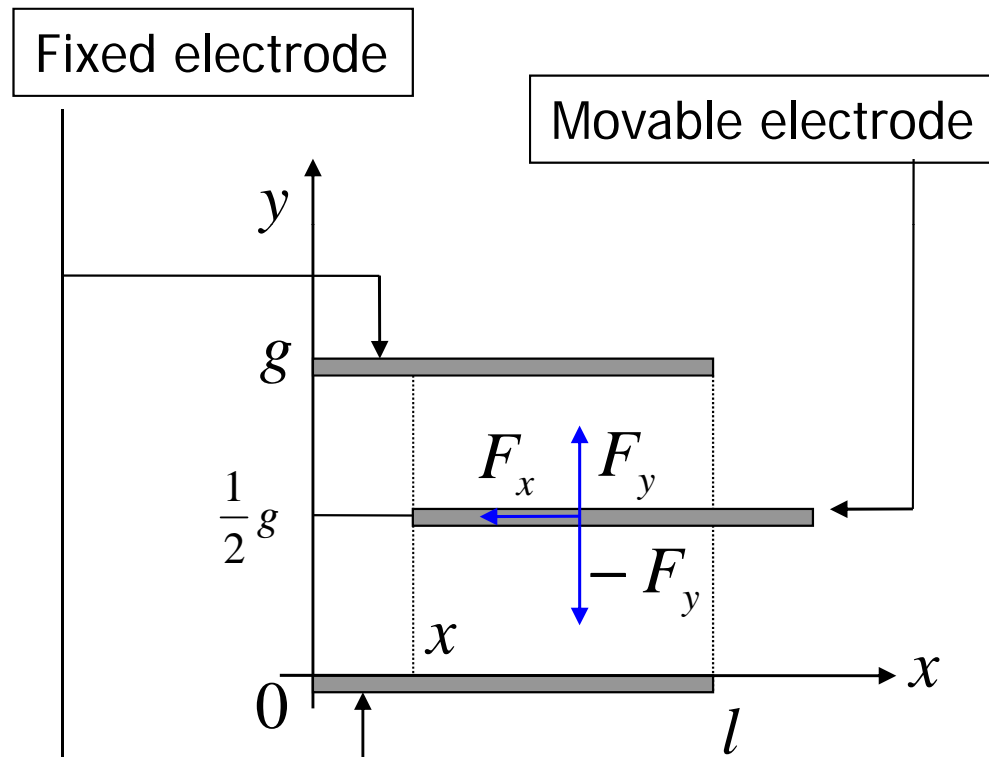
# Electrostatic force

## Comb-drive actuator Lateral movement



# Electrostatic force

## Comb-drive actuator Lateral movement



$$C = \frac{4\epsilon(l-x)b}{g-h}$$

$$U_E = \frac{2\epsilon(l-x)b}{g-h} V^2$$

$$F_x = \left( \frac{\partial U_E}{\partial x} \right)_{V=const} = -\frac{2\epsilon b}{g-h} V^2$$

h: thickness of electrode

- *Electrostatic force is independent of distance.*
- *It is proportional of number of electrodes.*

# Electrostatic force

## Comb-drive actuator \_ Vertical movement type

$$C = \frac{\varepsilon YZ}{g}, \quad U_E = \frac{\varepsilon YZ}{2g} V^2, \quad F_z = \left( \frac{\partial U_E}{\partial z} \right)_{V=\text{const}} = \frac{\varepsilon Y}{2g} V^2$$

## Rotational type

$$T_1 \propto \frac{1}{2} \frac{\partial C(\theta)}{\partial \theta} V^2$$

## Rotational type (Wobble motor)

$$T_2 \propto \frac{1}{2n} \frac{\partial C(\theta)}{\partial \theta} V^2, \quad n = \frac{R_i - r_0}{R_i}$$

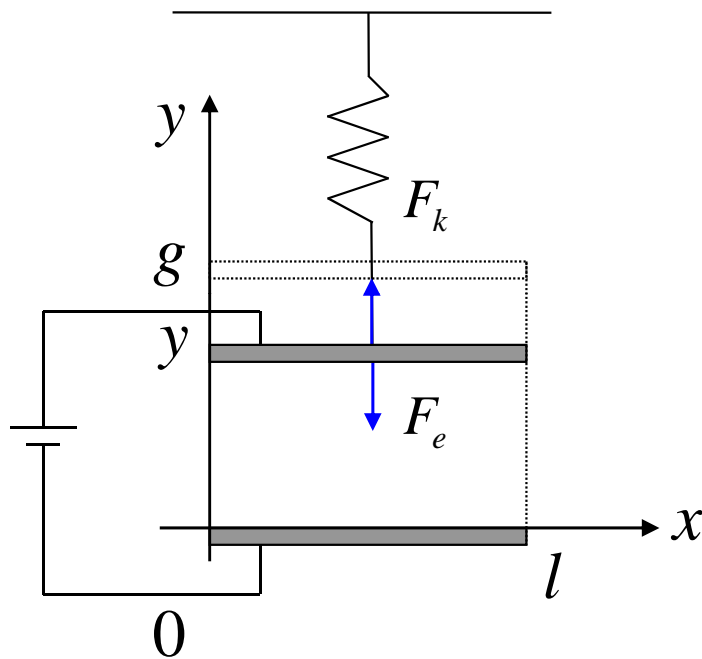
## Torsion bar type

$$C(\theta) = \int_{x_1}^{x_2} \frac{\varepsilon b}{\theta x} dx \rightarrow C(\theta) = \frac{\varepsilon b}{\theta} \log\left(\frac{h}{h - h\theta}\right), \quad U_E(\theta) = \frac{1}{2} \frac{\varepsilon b}{\theta} \log\left(\frac{h}{h - h\theta}\right) V^2$$



# Electrostatic force

## Pull-in phenomenon



Electrostatic force

$$F(y) = k(g - y) - \frac{\epsilon l b}{2y^2} V^2$$

Spring force

$$F(y) = -k(g - y)$$



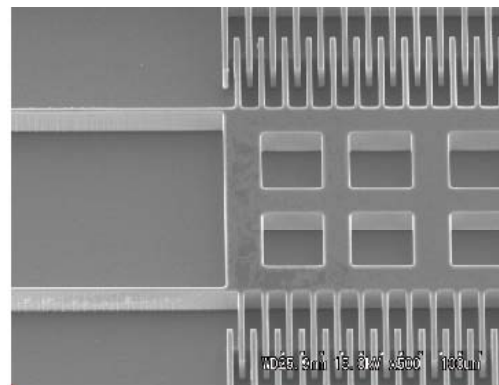
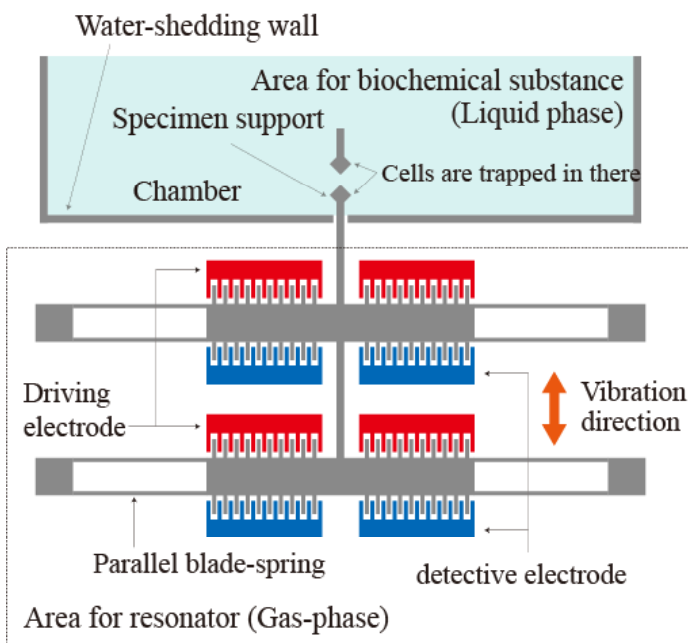
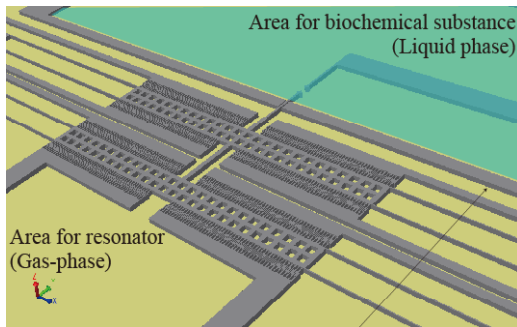
**Pull-in voltage**

$$V_{pull-in} = \left( \frac{8kg^3}{27\epsilon l b} \right)^{\frac{1}{2}}, \quad y_{pull-in} = \frac{2}{3} g$$

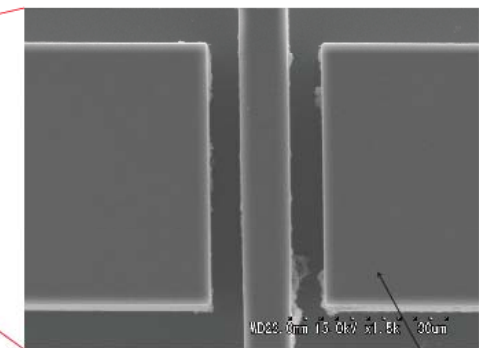
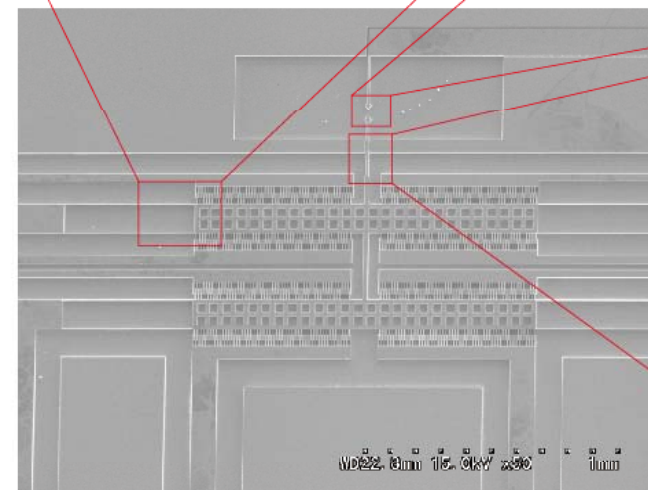
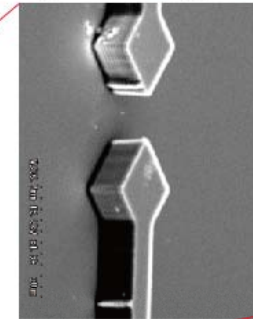


# Comb-drive-actuator

## Cell handling based on comb drive actuator

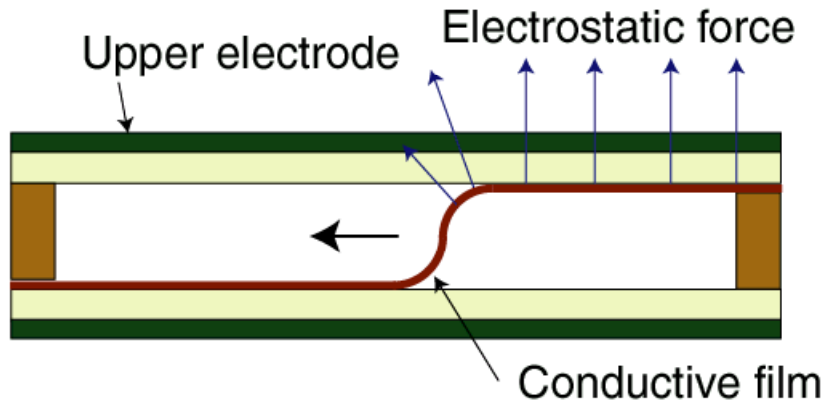


Specimen support

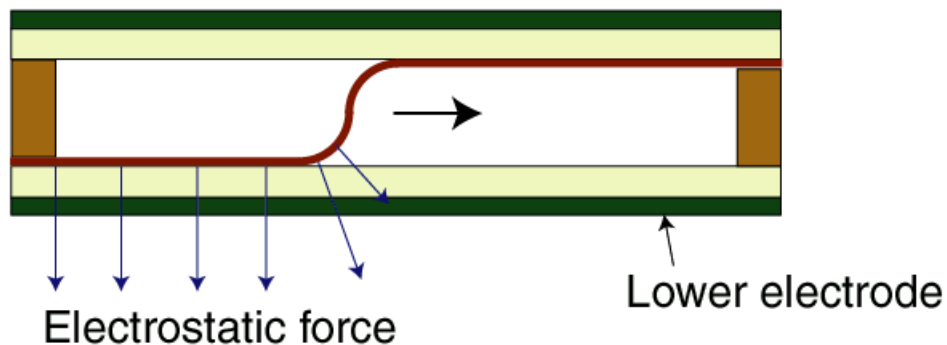


*H. Hida, et al., Tech. Dig. of IEEE MEMS, 2009, 912-915.*

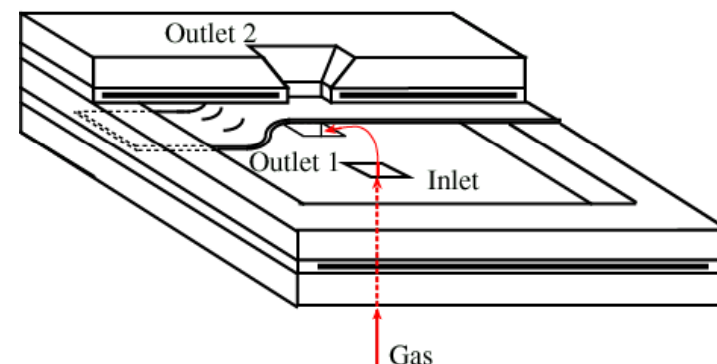
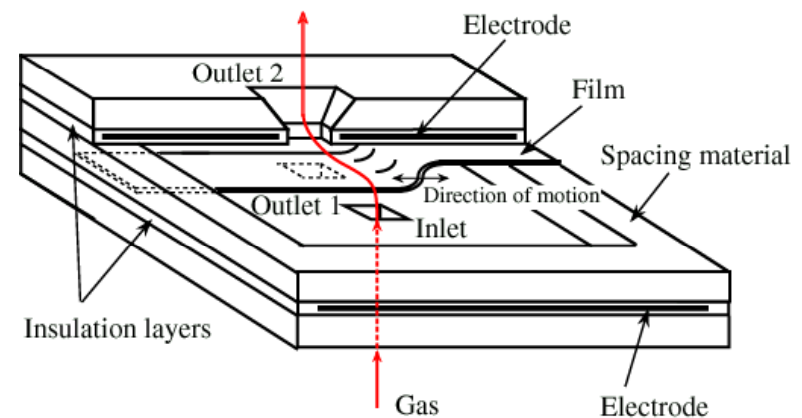
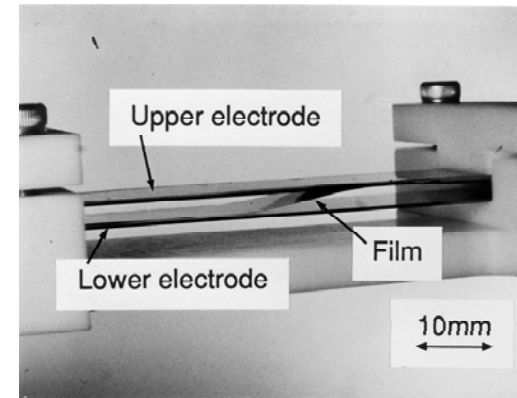
# S-shaped film actuator



(a) Voltage applied between upper electrode and conductive film.

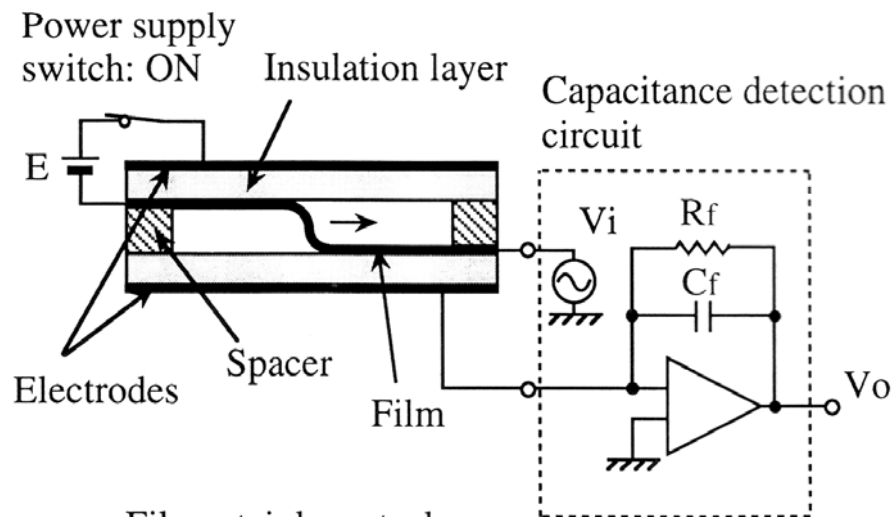


(b) Voltage applied between lower electrode and conductive film.



# S-shaped film actuator

## Characteristics in air



- Film: stainless steel  
thickness: 10  $\mu\text{m}$ , width: 12 mm
- Electrode: Si wafer  
thickness: 390  $\mu\text{m}$
- Insulation layer: SiO<sub>2</sub>  
thickness: 1.9  $\mu\text{m}$
- Spacer: glass plate  
thickness: 2.0 mm

図2 実験に用いたアクチュエータとフィルムの移動速度測定回路図

Fig. 2. Measurement scheme of film movement.

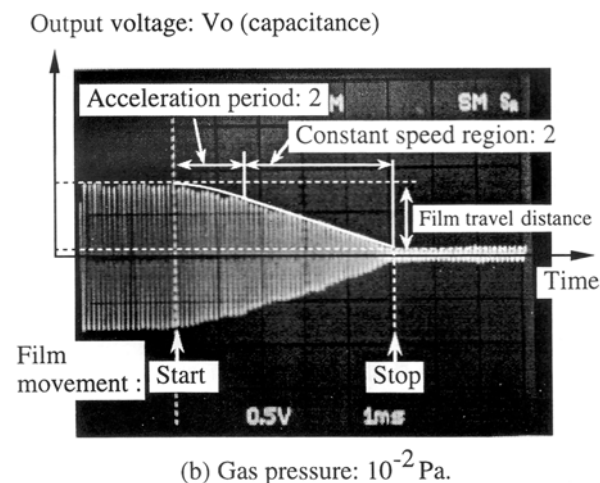
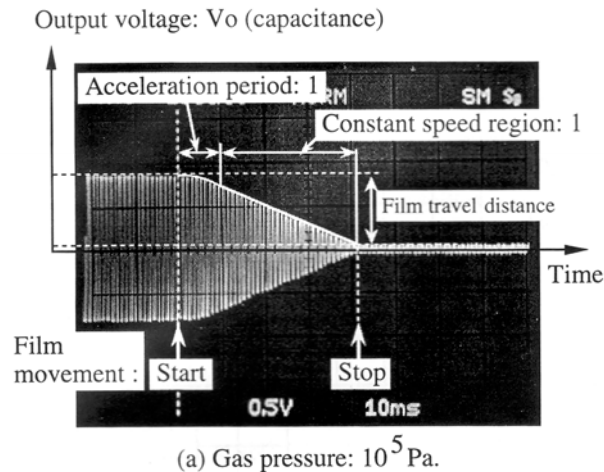


図4 フィルムの移動にともなう静電容量検出回路の出力波形の変化

Fig. 4. Measured capacitance change between film and lower electrode (applied voltage: 140 V).

M. Shikida., *T. IEE Japan*, 116-E, 1996, 395-401

# S-shaped film actuator

## Characteristics in air

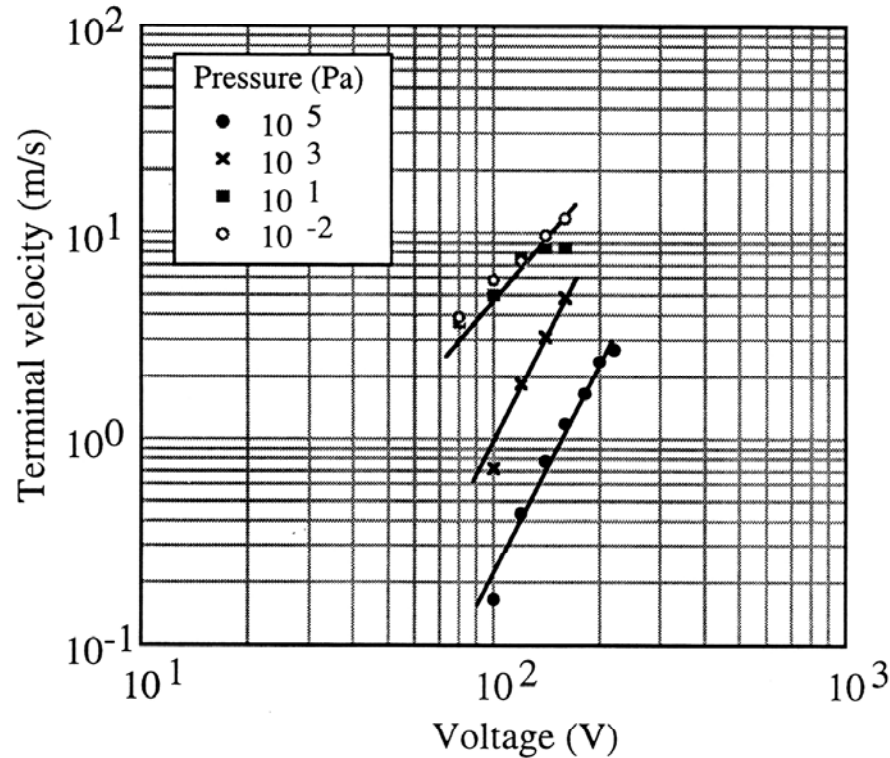


図7 フィルムの終端速度と駆動電圧との関係

Fig. 7. Relationship between terminal velocity of the S-bend and applied voltage.

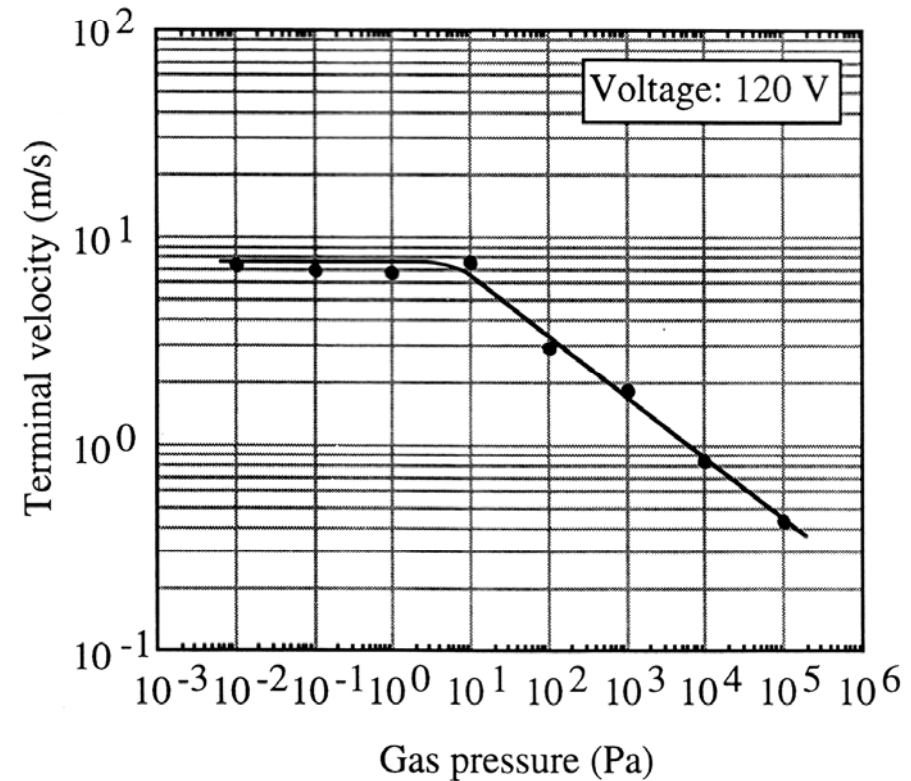


図8 雰囲気圧力がフィルムの終端速度に及ぼす影響

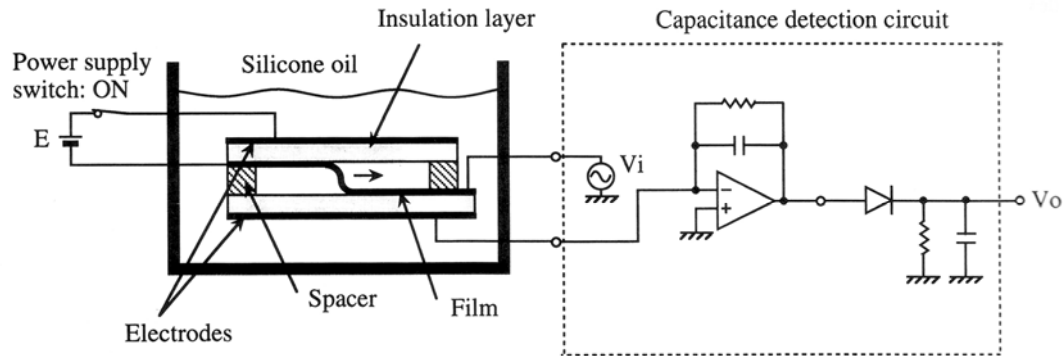
Fig. 8. Experimental results showing gas pressure dependence of terminal velocity.

M. Shikida., *T. IEE Japan*, 116-E, 1996, 395-401



# S-shaped film actuator

## Characteristics in liquid



- Film: stainless steel (thickness: 10-20  $\mu\text{m}$ , width: 12 mm)
- Electrode: Si wafer (thickness: 390  $\mu\text{m}$ )
- Insulation layer: SiO<sub>2</sub> (thickness: 1.9  $\mu\text{m}$ )
- Spacer: glass plate (thickness: 2.0 mm)

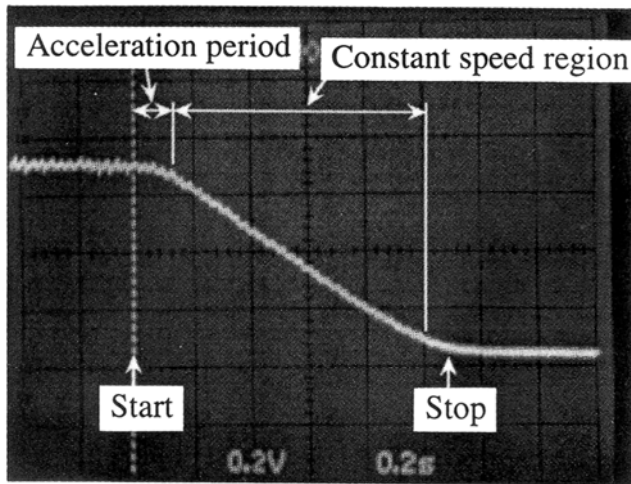
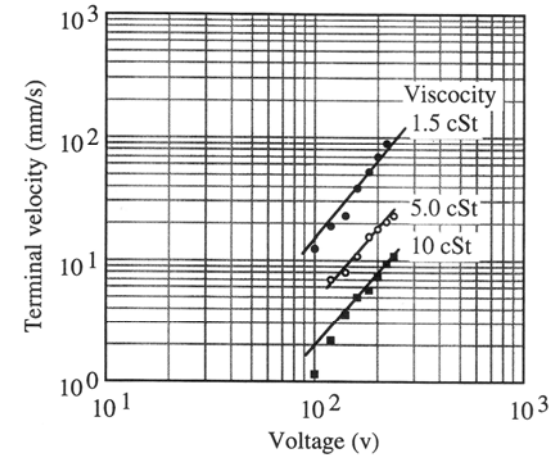


Fig. 3. Measured capacitance change between film and lower electrode (applied voltage: 180V).



(b) Viscosity influence

Fig. 4. Relationship between terminal velocity of the S-bend and applied voltage.

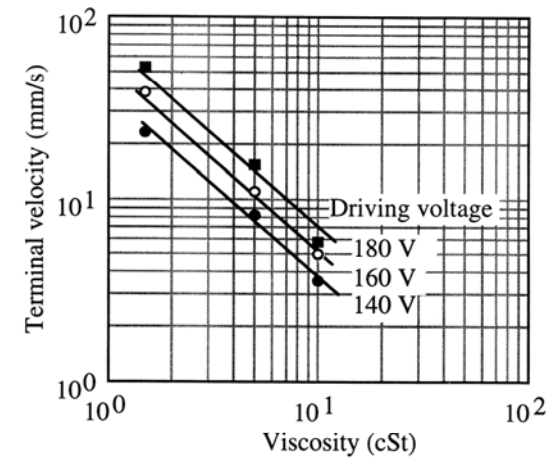


Fig. 5. Relationship between terminal velocity of the S-bend and viscosity.

*M. Shikida., T. IEE Japan, 117-E, 1997, 227-231*

# S-shaped film actuator

## Gas valve application

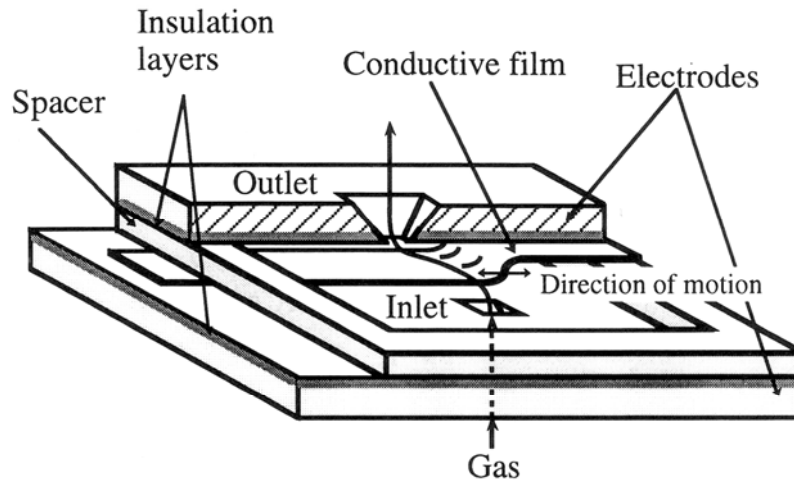


Fig. 1. Configuration of an electrostatically-driven two-port valve.

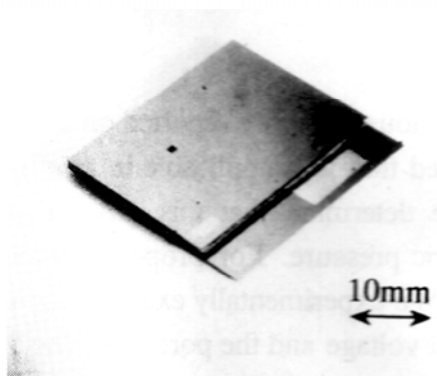


Fig. 2. Photograph of an experimental model.

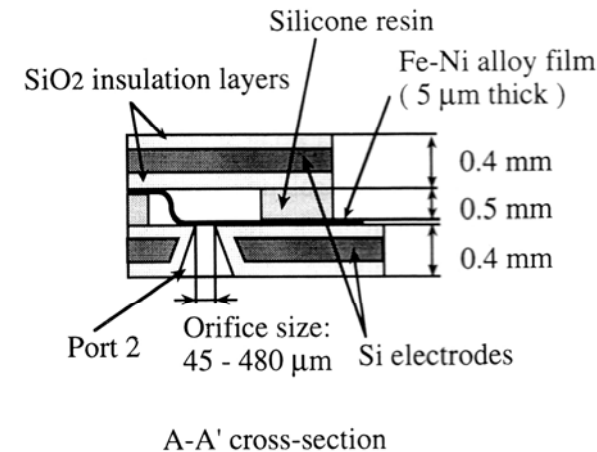
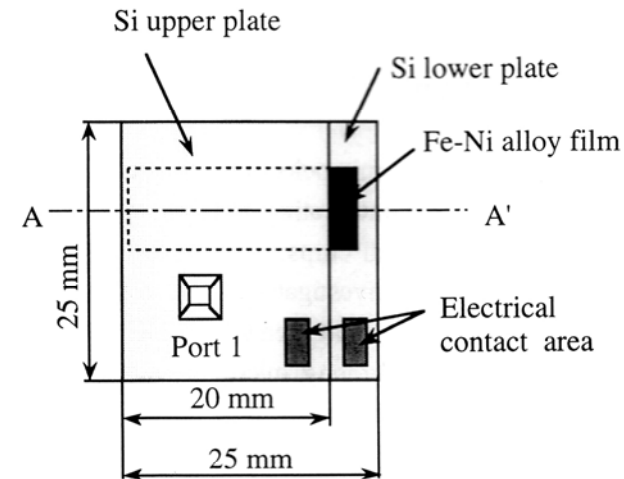


Fig. 3. Schematic top view and cross-section of an experimental model.

*M. Shikida., T. IEE Japan, 116-E, 1996, 219-223*

# S-shaped film actuator

## Gas valve application

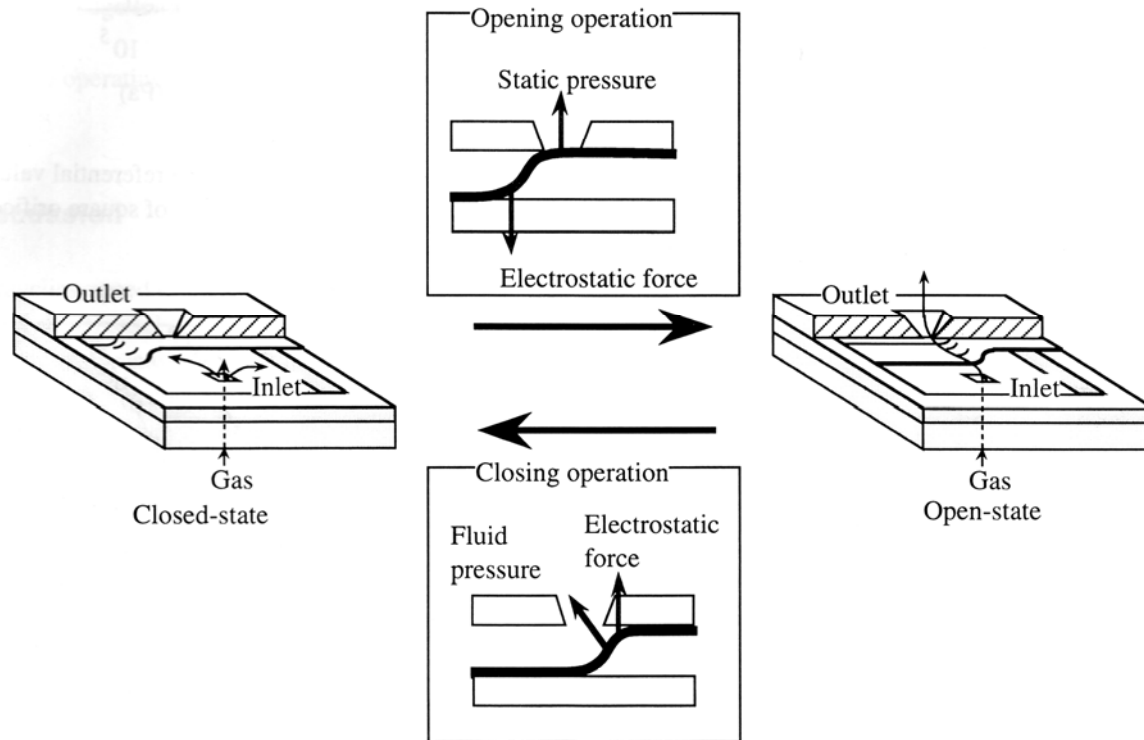


Fig. 5. Relationship between the electrostatic force and fluid pressure during valve operation.

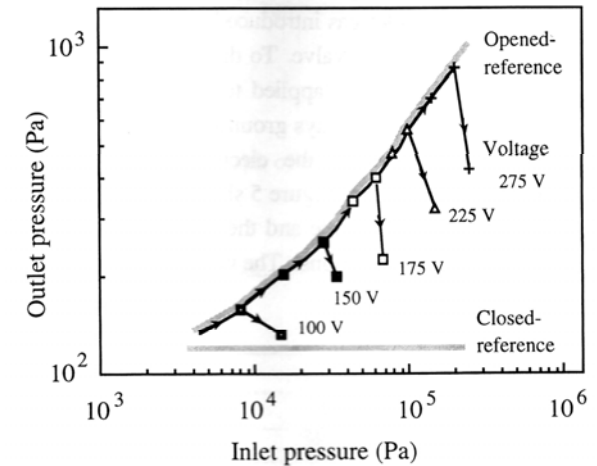


Fig. 7. Relationship between the referential values and outlet pressure (edge length of square orifice: 275 μm).

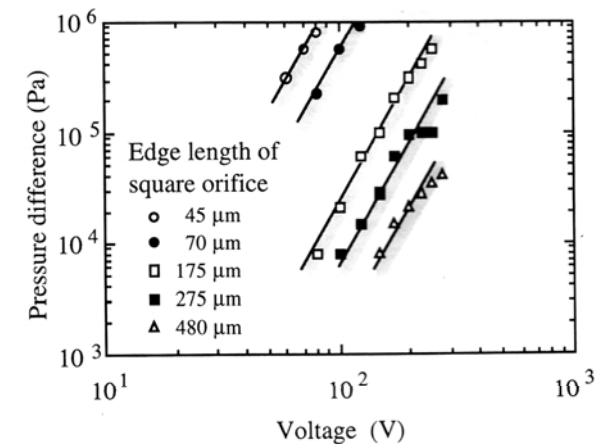


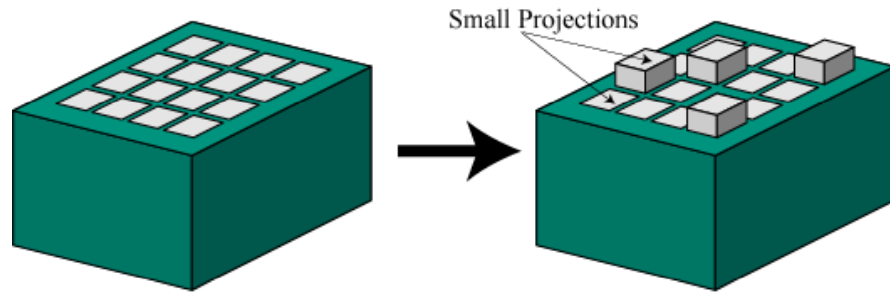
Fig. 8. Valve operating ranges (pressure difference vs. dc voltage) for various orifice sizes.

M. Shikida., *T. IEE Japan*, 116-E, 1996, 219-223



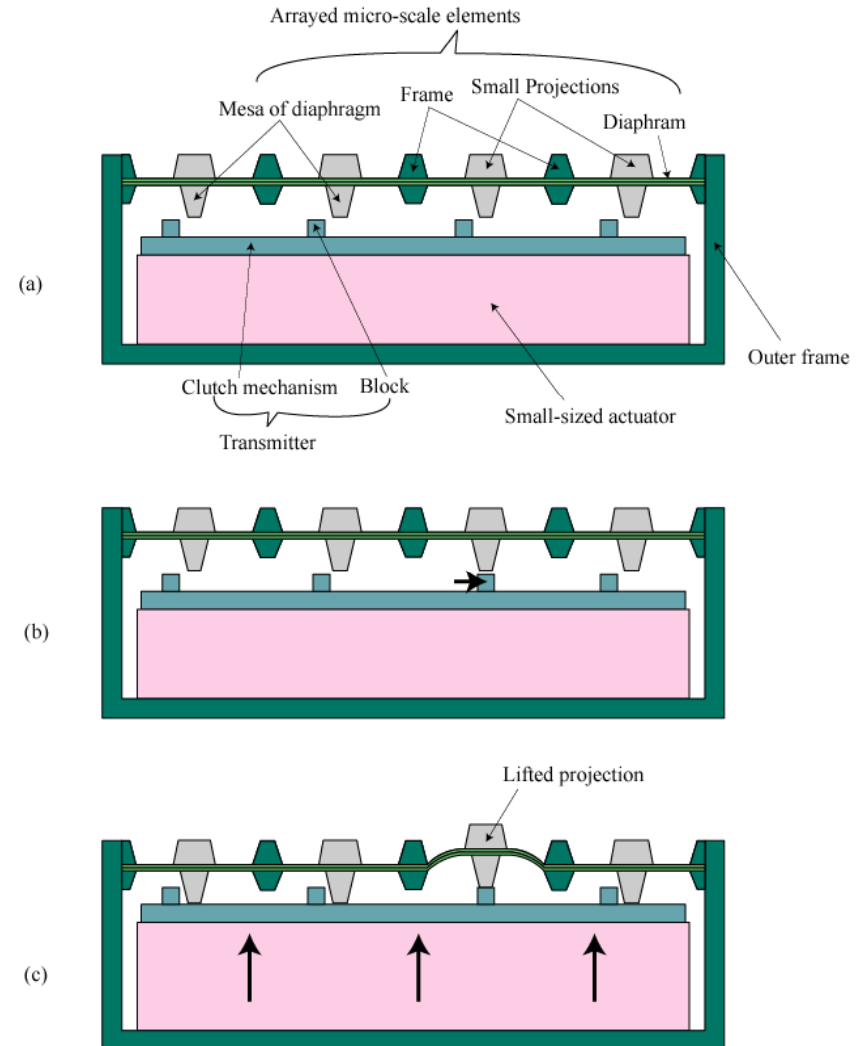
# Tulip-shaped actuator for haptic display

## Haptic display



Display  
Clutch produced by micro-actuator  
Stepping motor

## Operation principle



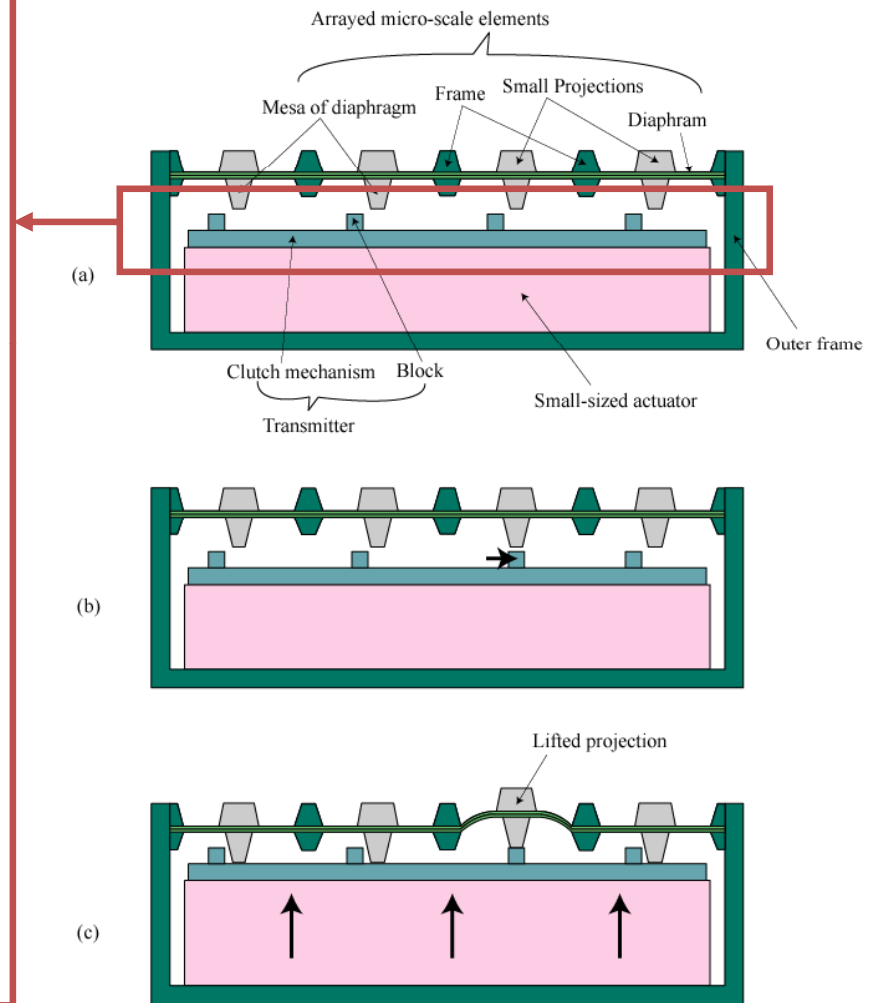
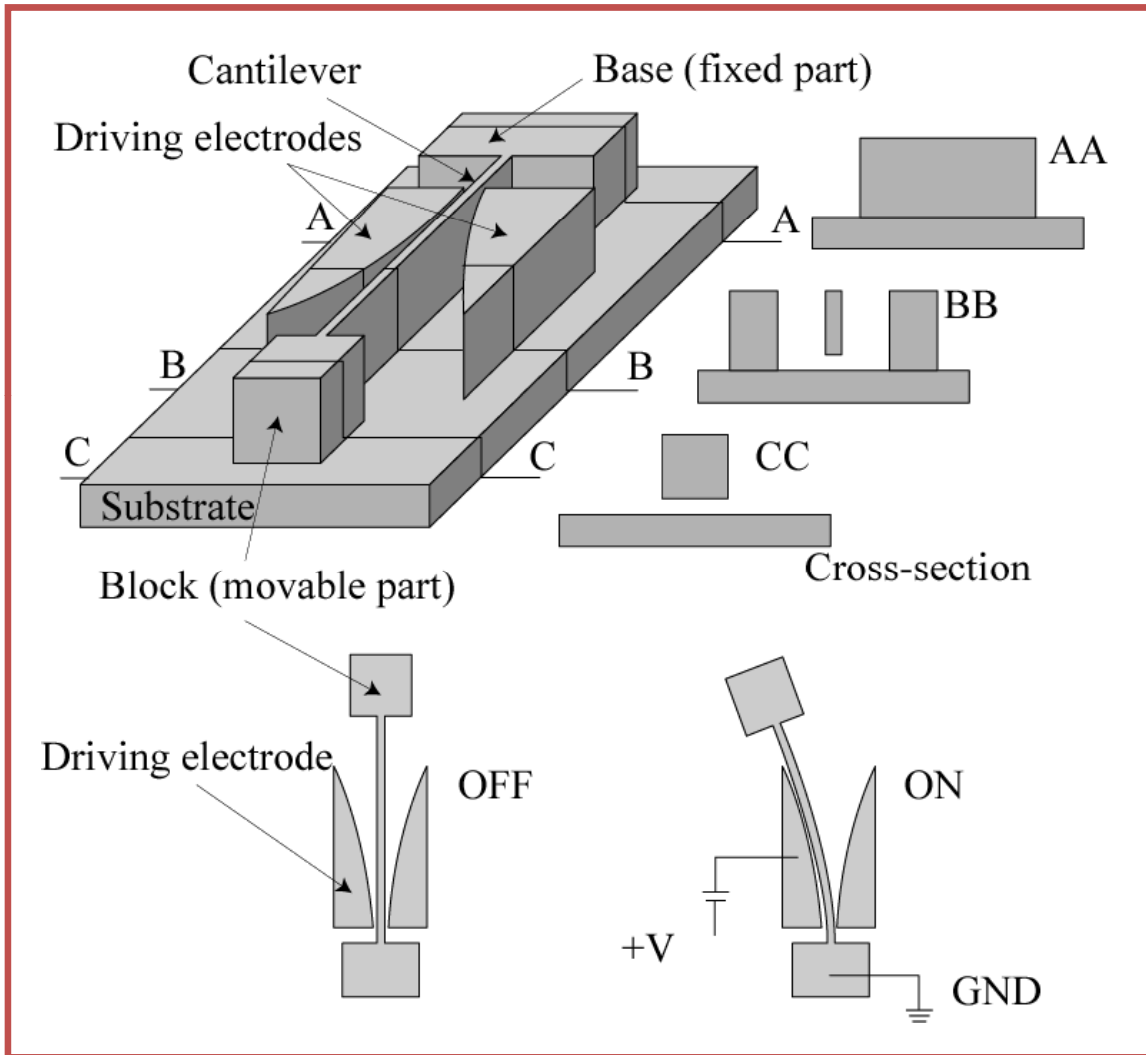
H. Sasaki, et al., *J. Micromech. Microeng.*, 16, 2006, 2673-2683.





# Tulip-shaped actuator for haptic display

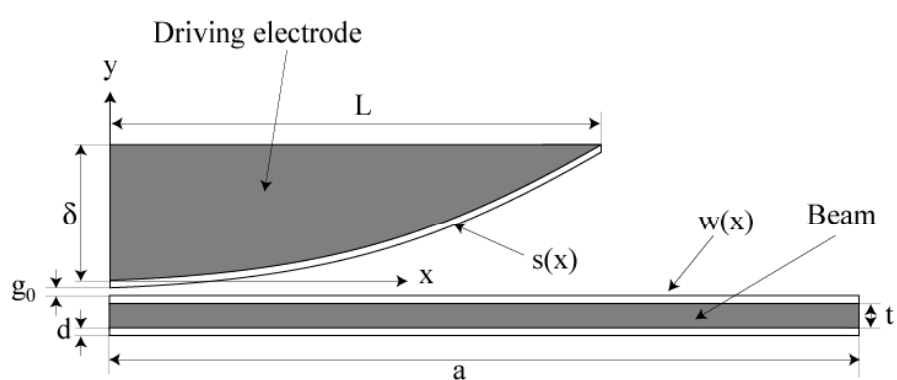
## Tulip-shaped actuator for clutch



H. Sasaki, et al., *J. Micromech. Microeng.*, 16, 2006, 2673-2683.

# Tulip-shaped actuator for haptic display

## Tulip-shaped actuator for clutch \_ analysis

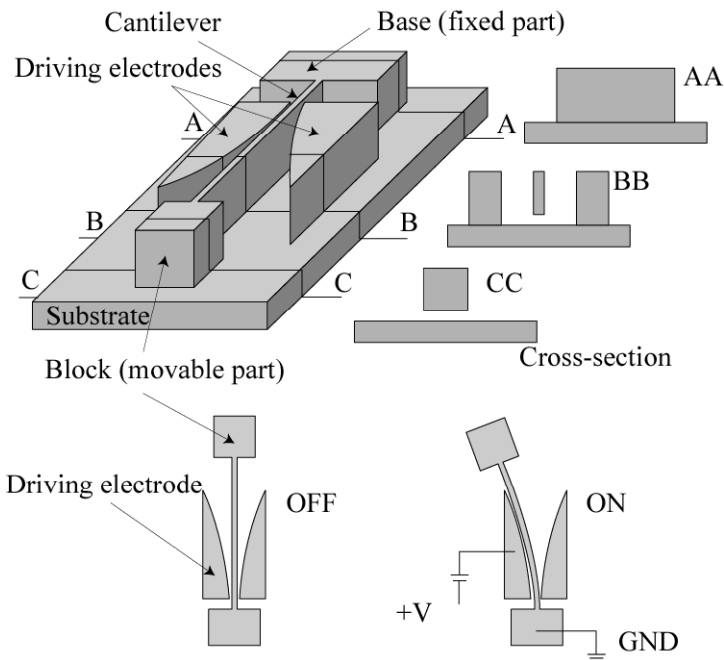


$$s(x) = \delta \left(\frac{x}{L}\right)^2$$

$$w(x) = cx^2 (6L^2 - 4Lx + x^2)$$

$$U_{el} = -\frac{1}{2} \int_0^L \frac{\epsilon_0 h V^2}{\frac{d}{\epsilon_r} + s(x) - w(x) + g_0} dx$$

$$U_b = \frac{1}{2} \int_0^L EI \left\{ \frac{d^2 w(x)}{dx^2} \right\}^2 dx$$

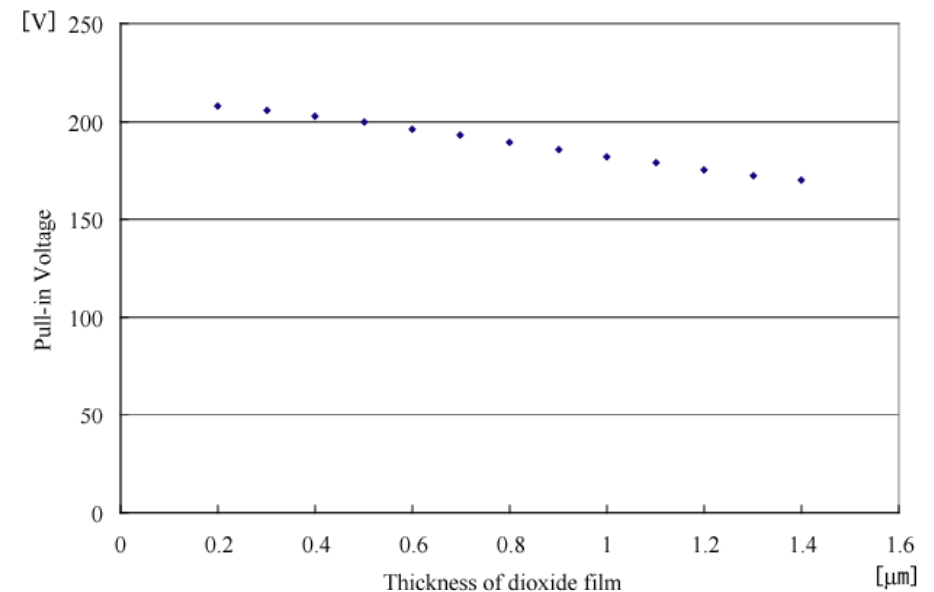
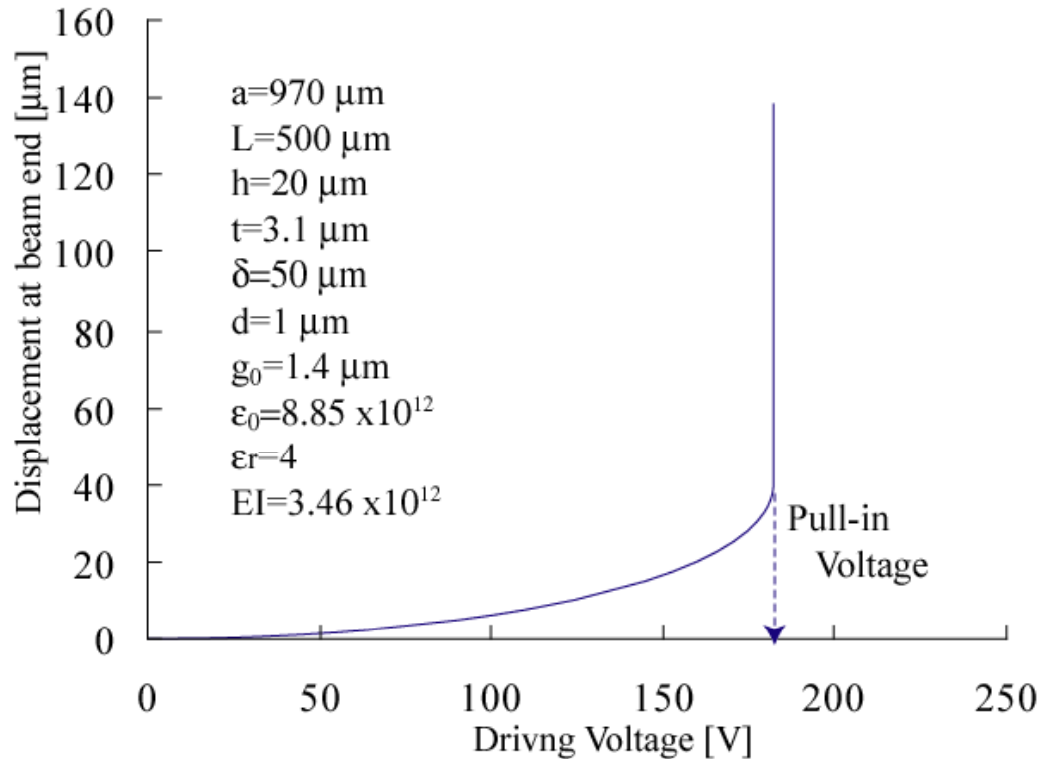
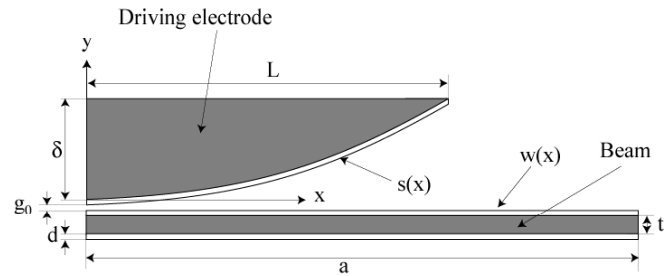


$$U = U_b + U_{el} = 0 \rightarrow V = \sqrt{\frac{EI \int_0^L EI \left[ \frac{d^2 w(x)}{dx^2} \right]^2 dx}{\epsilon_0 h \int_0^L \frac{1}{\frac{d}{\epsilon_r} + s(x) - w(x) + g_0} dx}}$$

H. Sasaki, et al., J. Micromech. Microeng., 16, 2006, 2673-2683.

# Tulip-shaped actuator for haptic display

## Tulip-shaped actuator for clutch \_ analysis

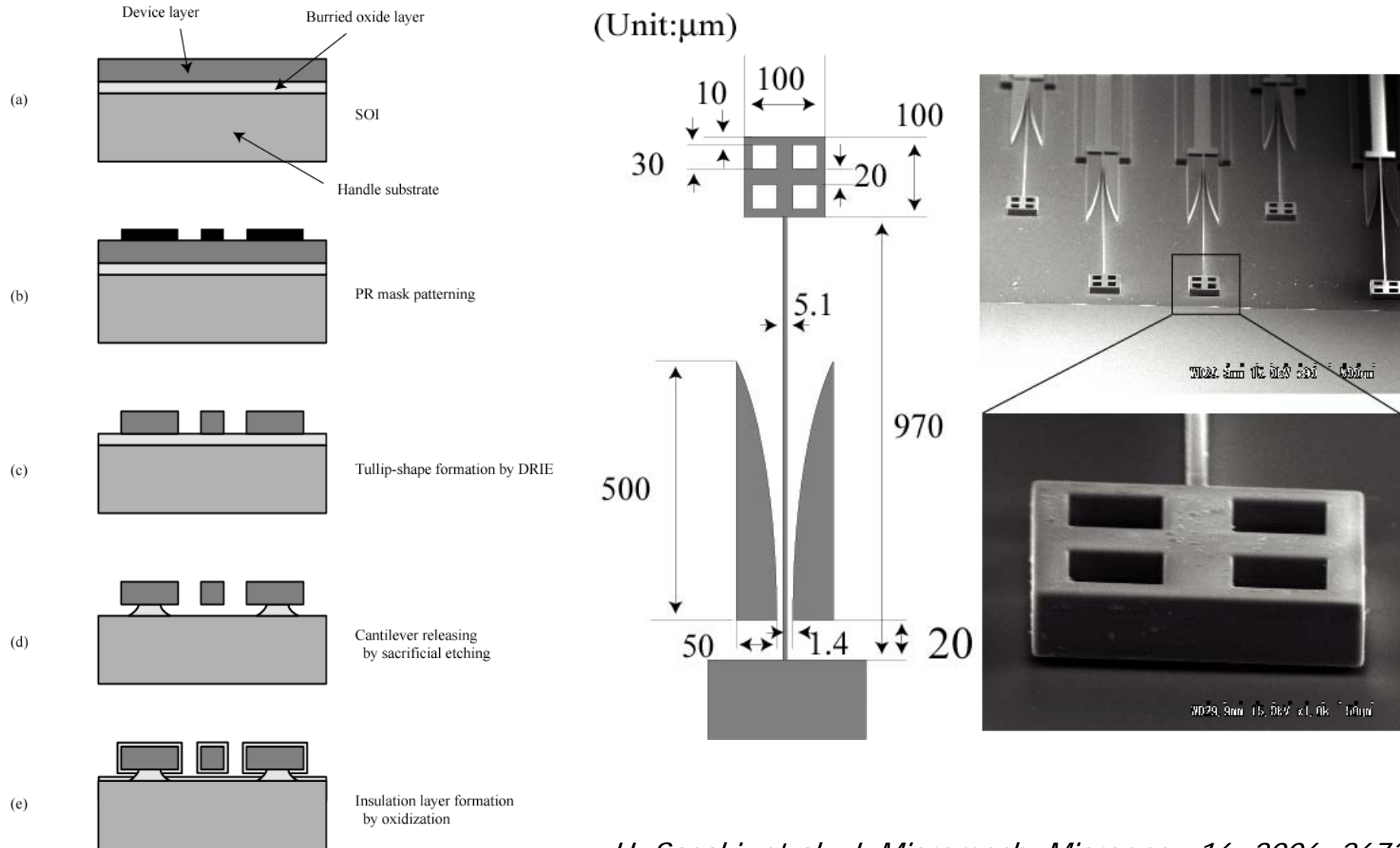


H. Sasaki, et al., *J. Micromech. Microeng.*, 16, 2006, 2673-2683.



# Tulip-shaped actuator for haptic display

## Tulip-shaped actuator for clutch \_ fabrication



H. Sasaki, et al., *J. Micromech. Microeng.*, 16, 2006, 2673-2683.

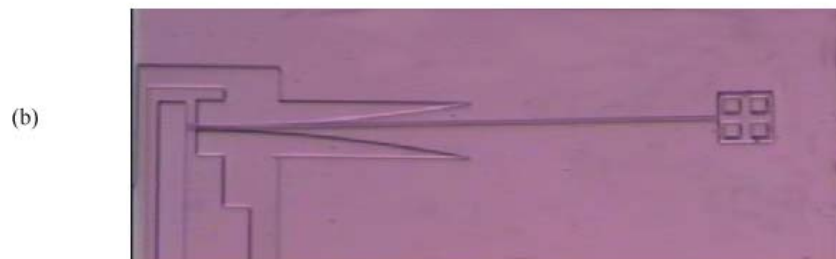


# Tulip-shaped actuator for haptic display

## Tulip-shaped actuator for clutch \_ operation



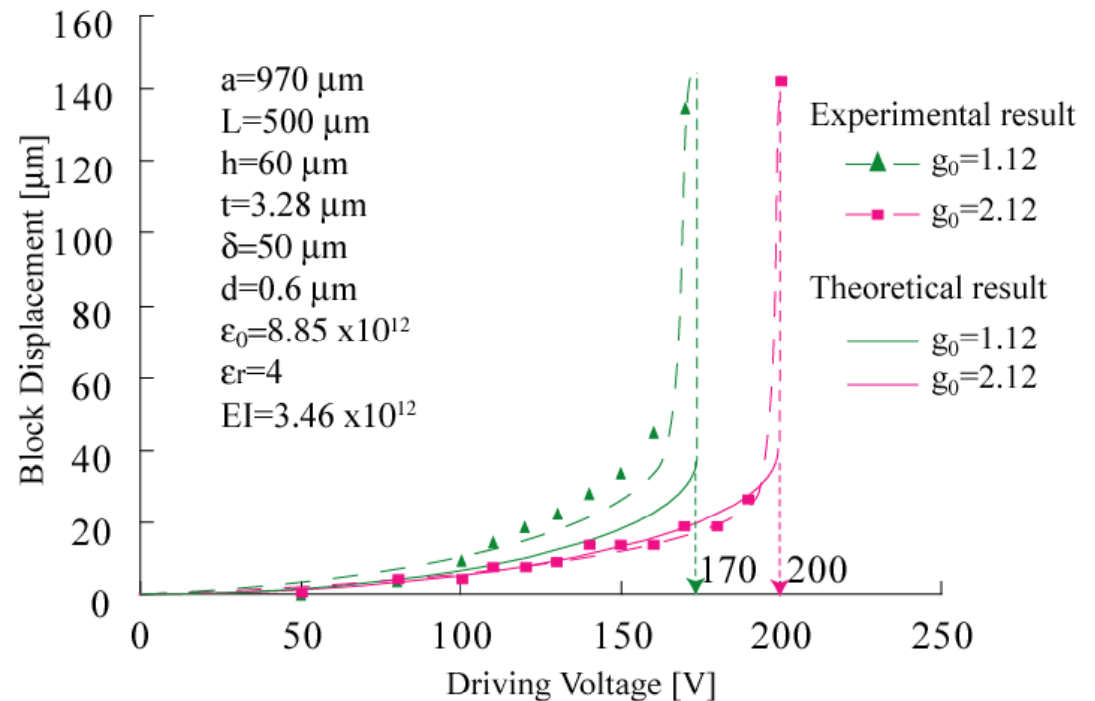
V=50V Deflection=0  $\mu\text{m}$



V=140V Deflection=32  $\mu\text{m}$



V=200V Deflection=138  $\mu\text{m}$



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# Tulip-shaped actuator for haptic display

## Assembled haptic display

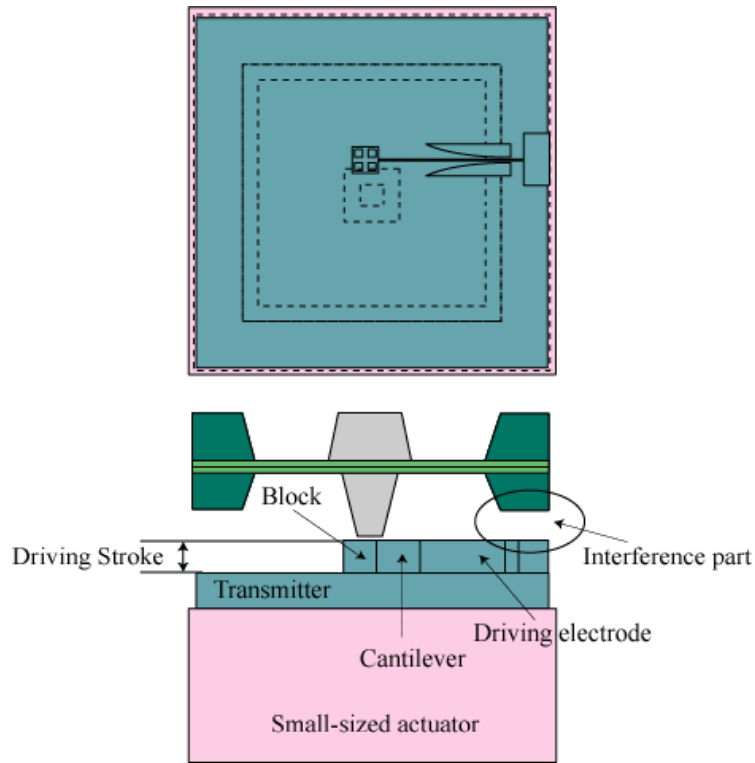
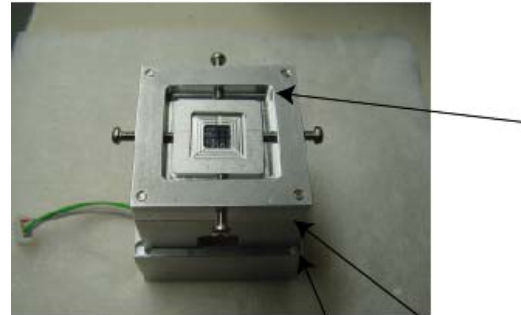
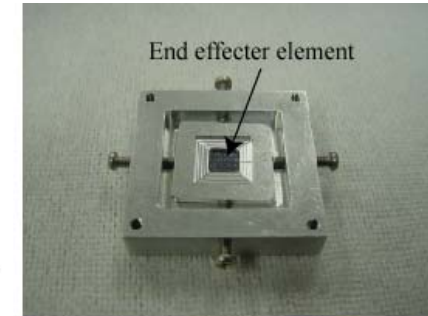


Image of a packaged element

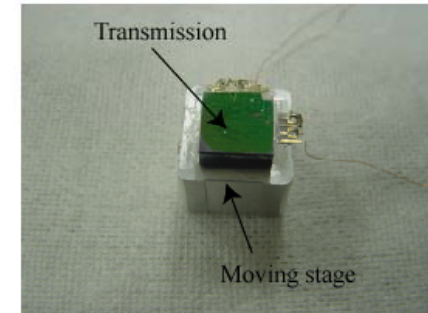
**Displacement: 20-60  $\mu\text{m}$**   
**Generative force: 600 mN**  
**Size: 30 mm x 30 mm x 40 mm**



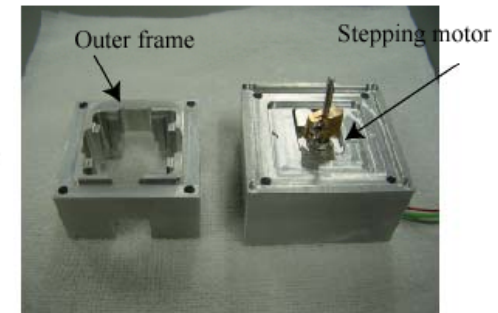
(a) Packaged Mechanical Power Transmission System



(b) Arrayed end effector element



(c) Tulip-shaped electrostatic clutch device and moving stage



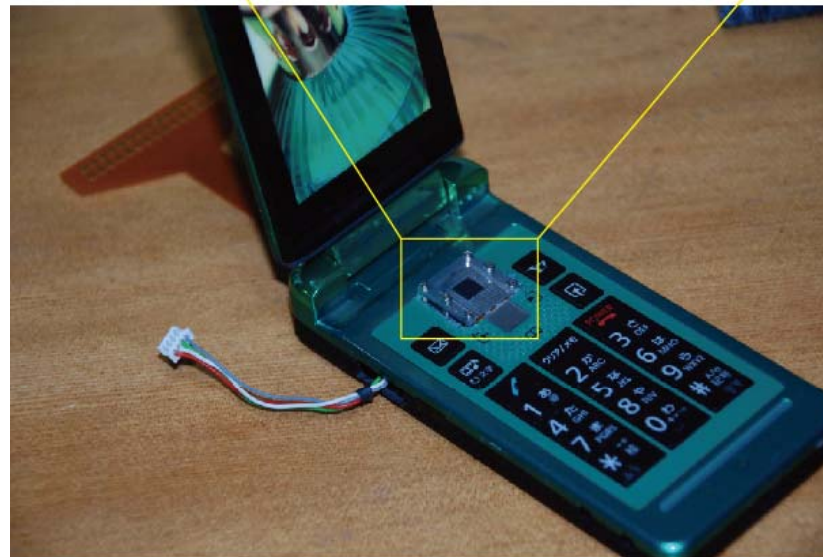
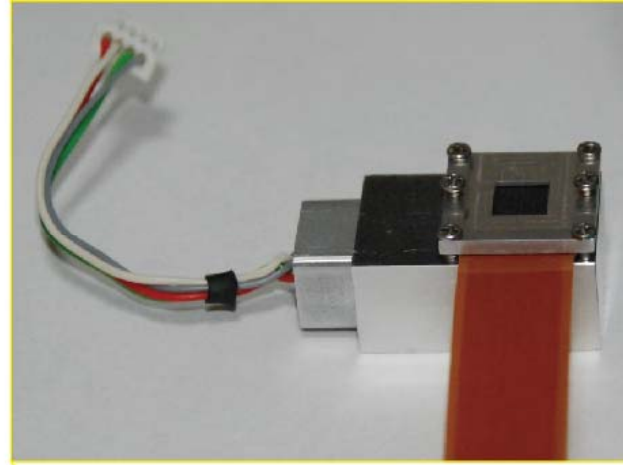
(d) Small-sized actuator

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## Demonstration of haptic display



# Summary

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- ✓ Overview of Micro-actuator
- ✓ Electrostatic force
  - ✓ Generative force
  - ✓ Comb-drive actuator
  - ✓ Pull-in phenomenon
- ✓ S-shaped film actuator for gas valve application
- ✓ Tulip-shaped actuator for haptic display

